THE DETERMINATION OF DISPLACEMENT FIELD AND NORMAL STRESSES IN MULTILAYERED SPHERICAL BODIES

Gönczi Dávid

PhD student

Institute of Applied Mechanics, University of Miskolc, Miskolc, Hungary

Abstract: The main objective of this paper is the calculation of the thermomechanical stresses and displacements in layered spherical bodies subjected to thermal and mechanical loadings. It is assumed that the temperature field and the displacement field depend only on the radial coordinate. The exact solution of one layer is used to get the analytical solution for the whole layered body by fitting the displacement and stress values on the boundary surfaces.

1. INTRODUCTION

This paper investigates a thermoelastic problem of a hollow layered spherical body. The geometry of the investigated body can be seen in Fig. 1, where the inner radius of the sphere is R_1 , the outer radius is R_{n+1} and n is the number of layers. The layers of the spherical structural component are assumed to be perfectly coupled and made of homogeneous, isotropic materials, furthermore a spherical coordinate system ($r\varphi \vartheta$) is used.

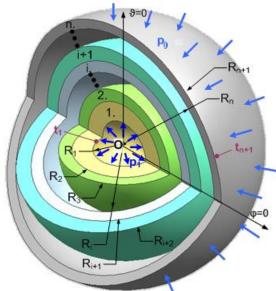


Figure 1. The three-dimensional sketch of the hollow layered sphere

First kind thermal boundary conditions are prescribed on the inner and outer spherical surfaces. These temperature values are given, they are assumed to be constant, non-time- dependent and denoted by t_1 and t_{n+1} . It follows that the temperature field T(r) is the function of the radial coordinate. The uniformly distributed mechanical loading exerted on the inner boundary surface is denoted by

 $f_1 = p_i$, while $-f_{n+1} = p_o$ is the pressure which acts on the outer curved boundary surface.

It is assumed that the radial stresses, the heatflow and the temperature are all continuous functions of the radial coordinate. Our aim is to determine the displacement field and normal stresses within the spherical component.

2. THE TEMPERATURE FIELD

At first we deal with the determination of the temperature field T=T(r). Figure 2 shows the cross section and the loadings of the *i*-th layer.

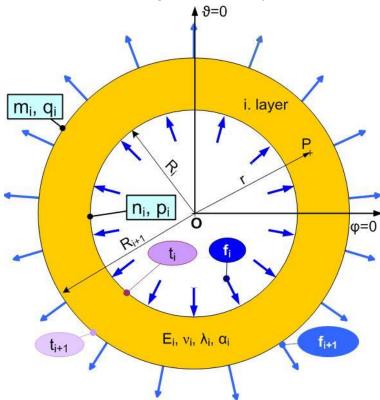


Figure 2. The cross section of the i-th layer of the sphere

We assume that the temperature field is a continuous function of the radial coordinate thus we have

$$t_{i+1} = T_i(R_{i+1}) = T_{i+1}(R_{i+1}); \quad T_i(r) = t_i - (t_{i+1} - t_i)(1 - \frac{R_i}{r})\frac{R_{i+1}}{R_i - R_{i+1}}, \quad i = 1, 2, ..., n, \quad (1)$$

where the temperature field of *i*-th spherical layer is denoted by $T_i(r)$. We consider the case when the radial heatflow is constant, the temperatures of the inner and outer boundary surfaces are given:

$$q_i(r) = \lambda_i (t_{i+1} - t_i) \frac{R_i R_{i+1}}{R_{i+1} - R_i} \frac{1}{r^2}, \quad i = 1, ..., n,$$
(2)

$$q_i(R_{i+1}) = q_{i+1}(R_{i+1}), \ i = 1, \dots, n-1.$$
(3)

where λ_i is the thermal conductivity. The surface temperature of the osculant layers are equal therefore we get the following equations [1]:

$$\left(-\lambda_{i}\frac{R_{i}R_{i+1}}{R_{i+1}-R_{i}}\right)t_{i} + \left(\lambda_{i}\frac{R_{i}R_{i+1}}{R_{i+1}-R_{i}} + \lambda_{i+1}\frac{R_{i+1}R_{i+2}}{R_{i+2}-R_{i+1}}\right)t_{i+1} + \left(-\lambda_{i+1}\frac{R_{i+1}R_{i+2}}{R_{i+2}-R_{i+1}}\right)t_{i+2} = 0.$$
(4)

3. SOLUTION OF THE STEADY-STATE THERMOELASTIC PROBLEM

The radial and tangential normal strains (ε_r , ε_{φ}) and the stress-strain relations of a homogeneous spherical body can be presented as [2,3]:

$$\varepsilon_r = \frac{\partial u}{\partial r}, \quad \varepsilon_{\varphi} = \varepsilon_{\varphi} = \frac{u}{r},$$
 (5)

$$\sigma_r(r) = \frac{E}{(1+\nu)(1-2\nu)} \Big[(1-\nu)\varepsilon_r + 2\nu\varepsilon_\varphi - (1+\nu)\alpha T(r) \Big], \tag{6}$$

$$\sigma_{\varphi}(r) = \sigma_{\varphi}(r) = \frac{E}{(1+\nu)(1-2\nu)} \Big[\nu \varepsilon_r + \varepsilon_{\varphi} - (1+\nu)\alpha T(r) \Big], \tag{7}$$

where u=u(r) is the radial displacement field, v is the Poisson ratio, E is the Young modulus, α is the coefficient of linear thermal expansion, $\sigma_r(r)$ is the radial normal stress and $\sigma_{\varphi}(r)$ is the tangential stress.

Let the displacement field for the *i*-th layer of the multilayered body be defined as:

$$u_{i}(r) = C_{i}r + \frac{D_{i}}{r^{2}} + U_{i}(r), \qquad (8)$$

where $U_i(r)$ has the following form [2,3]:

$$U_{i}(r) = \frac{1 + v_{i}}{1 - v_{i}} \alpha_{i} \frac{1}{r^{2}} \int_{R_{i}}^{r} T(\rho) \rho^{2} d\rho, \quad i = 1, ..., n.$$
(9)

With the combination of Eqs. (5), (6), (8) and (9) the expression of the radial stress for the *i*-th layer can be calcualted as

$$\sigma_{ri}(r) = C_i \frac{E_i}{1 - 2\nu} - D_i \frac{2E_i}{1 + \nu_i} \frac{1}{r^3} + S_i(r), \qquad (10)$$

$$S_{i}(r) = \frac{-2\alpha_{i}E_{i}}{1-\nu_{i}}\frac{1}{r^{3}}\int_{R_{i}}^{r}T(\rho)\rho^{2}d\rho, \ i = 1,...,n,$$
(11)

and the tangential stress is

$$\sigma_{\varphi i}(r) = C_i \frac{E_i}{1 - 2v_i} - D_i \frac{E_i}{1 + v_i} \frac{1}{r^3} - \frac{E_i \alpha_i T(r)}{1 - v_i} + \frac{E_i \alpha_i}{1 - v_i} \frac{1}{r^3} \int_{R_i}^{r} T(\rho) \rho^2 d\rho, \quad i = 1, ..., n.$$
(12)

The following discretized values of the displacement field and radial stresses will be used for the equations of the *i*-th layer:

$$u_i(R_i) = n_i, \quad u_i(R_{i+1}) = m_i, \quad \sigma_{ri}(R_i) = p_i, \quad \sigma_{ri}(R_{i+1}) = q_i.$$
 (13)

Using Eqs. (13), the unknown integration constants of Eqs. (8-12) can be calculated as

$$C_{i} = k_{1i}m_{i} + k_{2i}p_{i} - k_{1i}U_{i}(R_{i}) - k_{2i}S_{i}(R_{i}), \qquad (14)$$

$$D_{i} = k_{3i}m_{i} - k_{4i}p_{i} - k_{3i}U_{i}(R_{i}) + k_{4i}S_{i}(R_{i}), \qquad (15)$$

where

$$k_{1i} = \frac{2}{3R_i} \frac{1 - 2v_i}{1 - v_i}, \quad k_{2i} = \frac{(1 - 2v_i)(1 + v_i)}{3E_i(1 - v_i)}, \quad k_{3i} = \frac{R_i^2}{3} \frac{1 + v_i}{1 - v_i}, \quad k_{4i} = R_i^3 k_{2i}, \quad i = 1, \dots, n. (16)$$

From Eqs. (14-16) and (8-11) the expressions of the radial normal stress and radial displacement for the i-th layer can be obtained:

$$\sigma_{ri}(r) = K_{1i}(r)m_i + K_{2i}(r)p_i + K_{3i}(r)U_i(R_i) + K_{4i}(r)S_i(R_i) + S_i(r), \quad (17)$$

$$u_{ri}(r) = L_{1i}(r)m_i + L_{2i}(r)p_i + L_{3i}(r)U_i(R_i) + L_{4i}(r)S_i(R_i) + U_i(r),$$
(18)

where

$$K_{1i}(r) = \frac{E_i}{1 - 2v_i} k_{1i} - \frac{1}{r^3} \frac{2E_i}{1 + v_i} k_{3i}, \quad K_{2i}(r) = \frac{E_i}{1 - 2v_i} k_{2i} + \frac{1}{r^3} \frac{2E_i}{1 + v_i} k_{4i},$$

$$K_{3i}(r) = -\frac{E_i}{1 - 2v_i} k_{1i} + \frac{1}{r^3} \frac{2E_i}{1 + v_i} k_{3i}, \quad K_{4i}(r) = -\frac{E_i}{1 - 2v_i} k_{2i} - \frac{1}{r^3} \frac{2E_i}{1 + v_i} k_{4i},$$

$$L_{1i}(r) = rk_{1i} + \frac{k_{3i}}{r^2}, \quad L_{2i}(r) = rk_{2i} - \frac{k_{4i}}{r^2}, \quad L_{3i}(r) = -rk_{1i} - \frac{k_{3i}}{r^2}, \quad L_{4i}(r) = -rk_{2i} + \frac{k_{4i}}{r^2}.$$
(19)

Within the *i*-th layer of the spherical body the following matrix equation can be derived:

$$\begin{bmatrix} p_{i} \\ q_{i} \end{bmatrix} = \begin{bmatrix} -\frac{L_{1i}(R_{i+1})}{L_{2i}(R_{i+1})} & \frac{1}{L_{2i}(R_{i+1})} \\ K_{1i}(R_{i+1}) - K_{2i}(R_{i+1}) \frac{L_{1i}(R_{i+1})}{L_{2i}(R_{i+1})} & \frac{K_{2i}(R_{i+1})}{L_{2i}(R_{i+1})} \end{bmatrix} \begin{bmatrix} m_{i} \\ n_{i} \end{bmatrix} + \begin{bmatrix} -\frac{1}{L_{2i}(R_{i+1})} & 0 \\ -\frac{K_{2i}(R_{i+1})}{L_{2i}(R_{i+1})} & 1 \end{bmatrix} \begin{bmatrix} U_{i}(R_{i+1}) \\ S_{i}(R_{i+1}) \end{bmatrix} \\ + \begin{bmatrix} -\frac{L_{3i}(R_{i+1})}{L_{2i}(R_{i+1})} & -\frac{L_{4i}(R_{i+1})}{L_{2i}(R_{i+1})} \\ K_{3i}(R_{i+1}) - K_{2i}(R_{i+1}) \frac{L_{3i}(R_{i+1})}{L_{2i}(R_{i+1})} & K_{4i}(R_{i+1}) - K_{2i}(R_{i+1}) \frac{L_{4i}(R_{i+1})}{L_{2i}(R_{i+1})} \end{bmatrix} \begin{bmatrix} U_{i}(R_{i}) \\ S_{i}(R_{i}) \end{bmatrix} (21) \\ \begin{bmatrix} p_{i} \\ q_{i} \end{bmatrix} = \begin{bmatrix} G_{11}^{i} & G_{12}^{i} \\ G_{21}^{i} & G_{22}^{i} \end{bmatrix} \begin{bmatrix} m_{i} \\ n_{i} \end{bmatrix} + \begin{bmatrix} h_{1}^{i} \\ h_{2}^{i} \end{bmatrix}.$$

By the whole multilayered spherical body the following notations and fitting conditions will be used for the discretized values:

$$u_{i}(R_{i+1}) = u_{i+1}(R_{i+1}) = u_{i+1}, \ \sigma_{i}(R_{i+1}) = \sigma_{i+1}(R_{i+1}) = f_{i+1}, \ i = 1, \dots, n-1,$$
(23)

furthermore f_i and f_{n+1} are given. For the whole geometry the next system of equations can be derived for the displacement values \underline{u} as basic variables :

$$\begin{bmatrix} f_{1} \\ 0 \\ 0 \\ M \\ f_{n+1} \end{bmatrix} = \begin{bmatrix} G_{11}^{1} & G_{12}^{1} & 0 & 0 & 0 & L & 0 \\ G_{21}^{1} & G_{22}^{1} - G_{11}^{2} & -G_{13}^{2} & 0 & 0 & L & 0 \\ 0 & G_{21}^{2} & G_{22}^{2} - G_{11}^{3} & -G_{12}^{3} & 0 & L & 0 \\ M & & & & & & & \\ 0 & 0 & 0 & L & 0 & G_{21}^{n} & G_{22}^{n} \end{bmatrix} \begin{bmatrix} u_{1} \\ u_{2} \\ u_{3} \\ M \\ u_{n+1} \end{bmatrix} + \begin{bmatrix} h_{1}^{1} \\ h_{2}^{1} - h_{1}^{2} \\ h_{2}^{2} - h_{1}^{3} \\ M \\ h_{2}^{n} \end{bmatrix} . (24)$$
$$\underbrace{f = \underline{G}\underline{u} + \underline{h} . \qquad (25)$$

From Eqs. (24) and (25) the unknown displacement values $(u_i, i=1...n+1)$ can be calculated, and then, using Eq. (21) the radial normal stresses can be evaluated.

4. NUMERICAL EXAMPLES

We consider a three-layered spherical component. The first and third layers are made of a thermal insulation material, while the material of the second one is steel. For the numerical example the following data are used:

$$\begin{aligned} R_1 &= 0.15 \text{ m}; R_2 = 0.165 \text{ m}; R_3 = 0.195 \text{ m}; R_4 = 0.21 \text{ m}; E_1 = E_3 = 320 \text{ GPa}; E_2 = 211 \text{ GPa}; \\ v_1 &= v_3 = 0, 21; v_2 = 0, 3; \lambda_1 = \lambda_3 = 4 \frac{\text{W}}{\text{mK}}; \lambda_2 = 58 \frac{\text{W}}{\text{mK}}; \alpha_1 = \alpha_3 = 7.4 \cdot 10^{-6} \frac{1}{\text{K}}; \\ \alpha_2 &= 12 \cdot 10^{-6} \frac{1}{\text{K}}; p_i = 20 \text{ MPa}; p_o = 0 \text{ MPa}; t_1 = 523 \text{ K}; t_4 = 273 \text{ K}. \end{aligned}$$

Figs. 3-7 indicate the results of this problem, solved with *Maple 15*, in three cases.

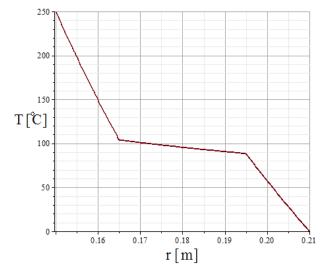


Figure 3. The temperature field

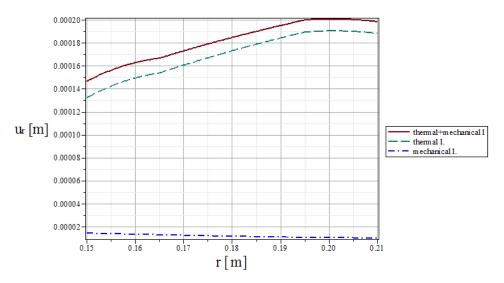


Figure 4. The radial displacement fields $u_r(r)$

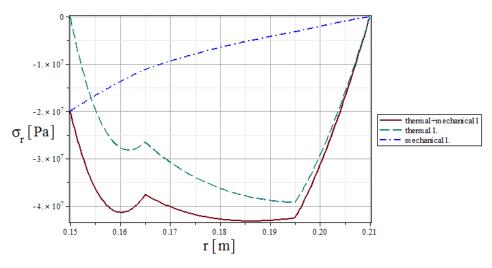


Figure 5. The radial stresses σ_r

In Figs. 4-6 the red solid lines are the results of the thermomechanical problem with $t_i=250^{\circ}$ C, $p_i=20$ MPa, the blue dashdot lines indicate the results for the case when there is only mechanical loading ($p_i=20$ MPa). The green dash lines illustrate the functions when there is only thermal loading ($t_i=250^{\circ}$ C, $p_i=0$ MPa)).

Fig. 7 indicates the von Mises equivalent stress of the thermomechanical problem (red solid line) and the black solid line is the curve for the case when all three layers are made of steel (the material of the middle layer in the original problem).

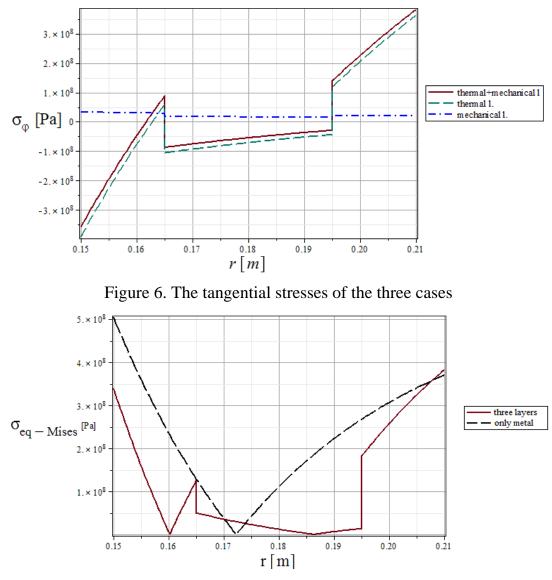


Figure 7. The equivalent stresses

5. CONCLUSIONS

The main objective of this paper was to present an analytical solution for the displacement field and the associated stresses in hollow layered spherical bodies subjected to mechanical and thermal loads. To solve this problem the Fourier's law of heat conduction and the exact solution for one layer were used, then the analytical solutions are derived via fitting the values of the displacement and stresses on the boundary surfaces of the layers. The developed solution can be utilized as Benchmark solutions for numerical methods to verify the accuracy of the considered numerical methods.

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